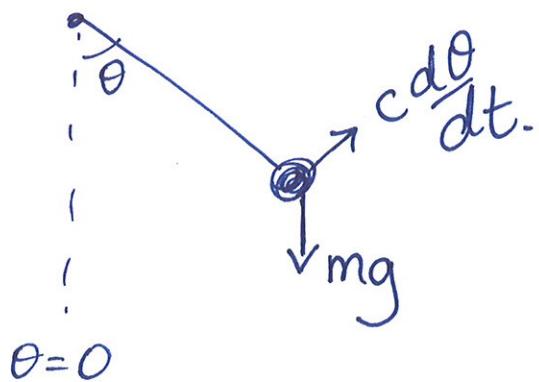


Undamped, Nonlinear Pendulum :



($c=0$ for now)
(undamped)

Equation :

$$\ddot{\theta} + \omega^2 \sin \theta = 0 \quad (\omega^2 = g/L)$$

AS a system:

Let $\theta = x$, $y = \dot{\theta}$:

$$\begin{cases} \dot{x} = y \\ \dot{y} = -\omega^2 \sin x \end{cases}$$

system
for
NL pend.

Critical points where

$$\dot{x} = 0 \text{ AND } \dot{y} = 0$$

ang. velocity

angular accel.

Are: $y = 0$ (angular vel.)
 $= 0$

AND. $X = n\pi$ for $n \in \mathbb{Z}$.

n even \rightarrow down pos.

$(n = 2m)$ $\leftarrow m \in \mathbb{Z}$

n odd \rightarrow up pos

$(n = 2m + 1)$

Anticipate:

- stable ~~ss~~ centre for n even.
- Saddle for n odd.

Compute:

First find Jacobian J :

$$J(x,y) = \begin{pmatrix} \frac{\partial F}{\partial x} & \frac{\partial F}{\partial y} \\ \frac{\partial G}{\partial x} & \frac{\partial G}{\partial y} \end{pmatrix} \text{ for } \begin{array}{l} \dot{x} = F(x,y) \\ \dot{y} = G(x,y) \end{array}$$

$$= \begin{pmatrix} 0 & 1 \\ -w^2 \cos x & 0 \end{pmatrix}, \text{ here.}$$

Linearize about. down
equil. ($x = 2m\pi$, $y = 0$)
 $\hookrightarrow x = 0$.

Let $\tilde{x} = 2m\pi + \tilde{x}$ (\tilde{x}, \tilde{y} "small")
 $y = 0 + \tilde{y}$

crit. point.

Then $\frac{d\tilde{x}}{dt} = \frac{d}{dt}(2m\pi + \tilde{x}) = J(2m\pi, 0)(\begin{pmatrix} \tilde{x} \\ \tilde{y} \end{pmatrix})$
 $= \begin{pmatrix} 0 & 1 \\ -w^2 & 0 \end{pmatrix} \begin{pmatrix} \tilde{x} \\ \tilde{y} \end{pmatrix}$

Type and stability of critical

point from e-vals/e-veets
of $J(2m\pi, 0) = \begin{pmatrix} 0 & 1 \\ -\omega^2 & 0 \end{pmatrix}$.

eigenvalues are $\pm i\omega$

corresp. eigenveets $\begin{pmatrix} 1 \\ \pm i\omega \end{pmatrix}$.

Since e-vals are purely imaginary, $(2m\pi, 0)$ (or $(0, 0), (2\pi, 0)$, etc)

is are stable centres

$(m \in \mathbb{Z})$.

Linearize about up equil.

$(x = (2m+1)\pi, y=0)$

$\hookrightarrow x = -\pi, \pi, 3\pi, \dots$

$$x = (2m+1)\pi + \tilde{x} \quad (\tilde{x}, \tilde{y} \text{ small})$$

$$y = 0 + \tilde{y}$$

obtain $\frac{d}{dt} \begin{pmatrix} \tilde{x} \\ \tilde{y} \end{pmatrix} = J((2m+1)\pi, 0) \begin{pmatrix} \tilde{x} \\ \tilde{y} \end{pmatrix}$

↑ Jacobian evaluated at crit. point.

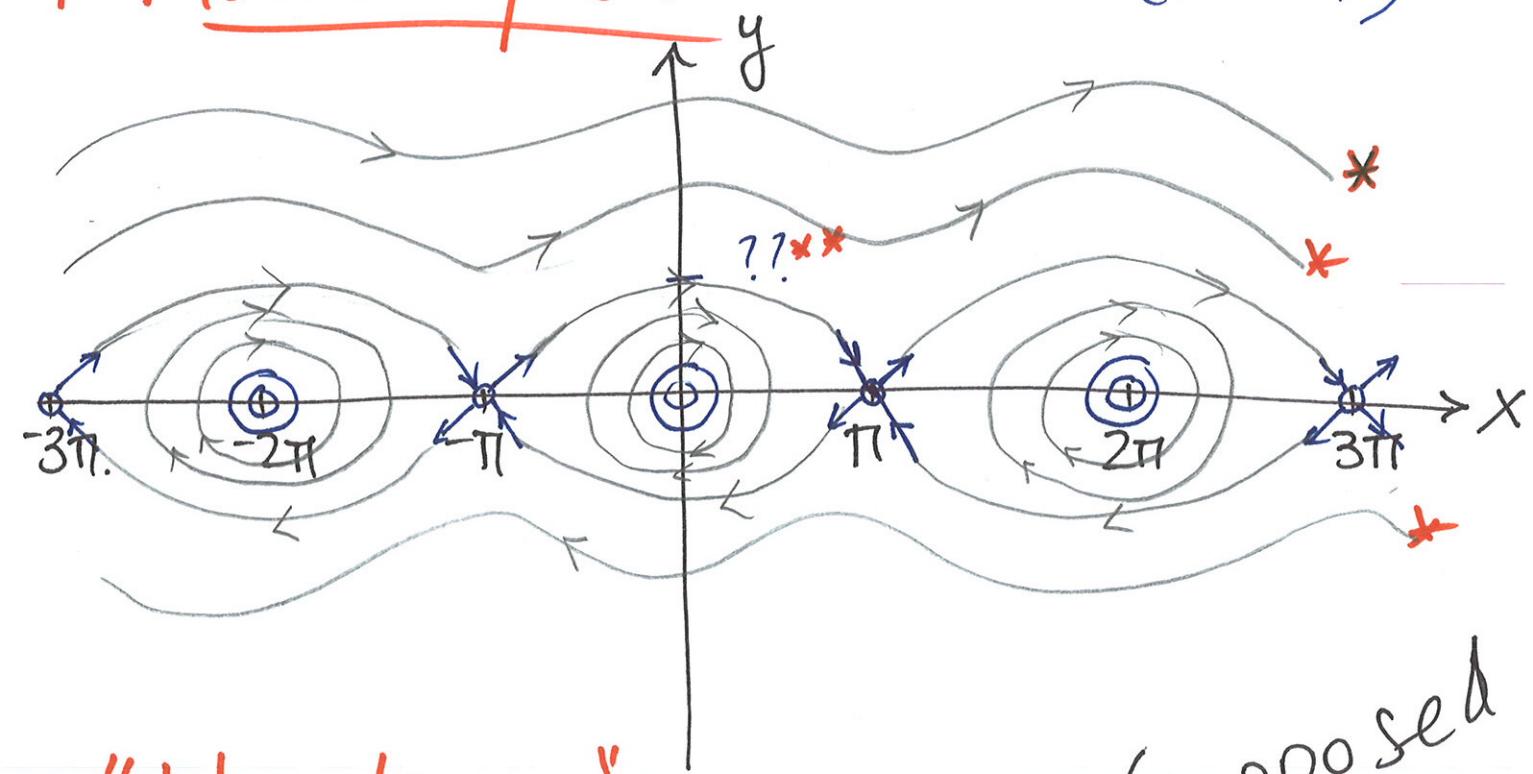
- type & stab of crit. point from e-vats/e-vecs of $J((2m+1)\pi, 0)$.

$$J((2m+1)\pi, 0) = \begin{pmatrix} 0 & 1 \\ \omega^2 & 0 \end{pmatrix}.$$

eigenvalues are $r_{\pm} = \pm \omega$
with eigenvects $\begin{pmatrix} 1 \\ \pm \omega \end{pmatrix}$.

e-vals are real & opposite sign \rightsquigarrow saddle point.

Phase portrait: ($\omega = 1$)



* "librations"

initial vel. so high
pendulum goes over
the top.

(supposed
to look
the same)

** use conservation of energy to
show this value $\theta_{\max} = \dot{\theta}_{\max} = 2\omega$.
 $(=y)$.

Powerful theory! We used linearization to get an approx. picture of the full NL system.

Compare with Linear
(simple) pendulum
→ show approximation.

P PERIOD of oscillations.

Linear pendulum

$$\ddot{\theta} + \omega^2 \theta = 0$$

Period was $\frac{2\pi}{\omega}$ ($= 2\pi\sqrt{\frac{L}{g}}$)

NL pendulum:

conservation form of ODE:

$$\frac{\dot{\theta}^2}{2} - \omega^2 \cos\theta = C$$

Assume you drop pendulum
($\dot{\theta}(0) = 0$) at angle θ_0 .

$$\Rightarrow C = -\omega^2 \cos\theta_0$$

$$\frac{\dot{\theta}^2}{2} - \omega^2 \cos\theta = -\omega^2 \cos\theta_0$$

(for pend. dropped at θ_0)

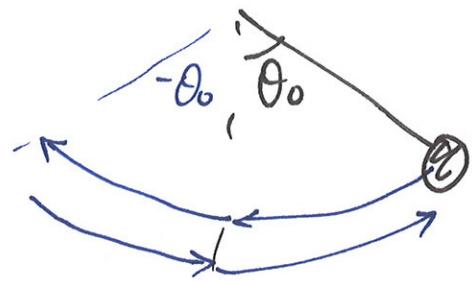
Solve for $\dot{\theta}$:

$$\frac{d\theta}{dt} = -\sqrt{2\omega^2 (\cos\theta - \cos\theta_0)}$$

$C - b/c$ towards equil.

$$dt = \frac{-d\theta}{\sqrt{2\omega^2 (\cos\theta - \cos\theta_0)}}$$

Period



$$\int_0^{\pi/4} dt = \frac{-1}{\sqrt{2w^2}} \int_{\theta_0}^0 \frac{d\theta}{\sqrt{\cos\theta - \cos\theta_0}}$$

$$T = \frac{+4}{\sqrt{2w^2}} \int_0^{\theta_0} \frac{d\theta}{\sqrt{\cos\theta - \cos\theta_0}}$$

(can write as
elliptic int.)

period
depends on amplitude
 θ_0 . (different)

\Rightarrow truly are approximating
BUT phase portrait is
very helpful.